

## GUJARAT TECHNOLOGICAL UNIVERSITY

BE- SEMESTER-VII EXAMINATION – WINTER 2025

Subject Code:3170514

Date:28-11-2025

Subject Name:Mechanical Design of Process equipments

Time:10:30 AM TO 01:00 PM

Total Marks:70

**Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.
4. Simple and non-programmable scientific calculators are allowed

- Q.1** (a) What is stiffening rings? When can it be used in pressure vessel design? **03**  
(b) Write a technical note on “Radiography test” for pressure vessel. **04**  
(c) What is flange joint? With neat sketch explain the different types of standard flanges used in industries. **07**
- Q.2** (a) Differentiate between fatigue and creep. **03**  
(b) Name the methods used to calculate the shell thickness under external pressure and discuss any one of them with necessary equations and design steps. **04**  
(c) A Reactor (ID = 800 mm) with hemispherical head at the bottom. Inside working pressure is 75 kgf/cm<sup>2</sup> (g) & working temperature is 70 °C. Reactor is covered with plain jacket such that 75% length of shell & bottom hemispherical head is covered with jacket. Cooling water is circulated inside the jacket by pumping with a centrifugal pump having a shut off discharge pressure 6.0 kgf/cm<sup>2</sup>(g). The hemispherical head is fabricated from SA-516 Grade 70. The maximum allowable stress at design temperature is 610 kgf/cm<sup>2</sup>. Modulus of Elasticity of plate material (E) =  $193 \times 10^3$  N/mm<sup>2</sup>. Poisson’s ratio ( $\mu$ ) = 0.3,  $\rho$  = 7.83 g/cm<sup>3</sup>. Joint efficiency (J) = 0.85. Take 3 mm corrosion allowance. Find:  
(i) Thickness of the head and (ii) weight of the fabricated head. **07**
- OR**
- (c) Find out the thickness of shell & jacket for the (i) reactor with plain jacket and (ii) reactor with channel jacket. The data are given as follow. **07**  
Inside diameter of shell - 1500 mm, Inside diameter of jacket - 1600 mm, Shell length - 1500 mm, Diameter of half coil - 75 mm, Width of channel jacket - 75 mm, Internal design pressure for shell - 4 Kgf/cm<sup>2</sup>, Internal design pressure for jacket - 3 Kgf/cm<sup>2</sup>, Design temperature for shell and jacket - 150 C, Material of shell - SA 316 Gr 70, Maximum allowable stress at design temperature - 980 Kgf/cm<sup>2</sup>, Modulus of elasticity -  $19 \times 10^5$  kgf/cm<sup>2</sup>, Poisson’s ratio - 0.3, Joint efficiency - 0.85. Take corrosion allowance as 1.5 to 3 mm.
- Q.3** (a) State the uses of propeller, paddle and turbine agitator. **03**  
(b) Discuss the purpose of providing reinforcement pad for nozzle. **04**  
(c) With neat sketch explain the various types of jackets and discuss the design of channel jacket and half coil jacket for reaction vessel. **07**

**OR**

- Q.3** Design a bracket support for reaction vessel based on given data. Brackets are welded with outside surface of the reactor shell. OD of reactor shell = 1052.7 mm, Thickness of the shell = 6.35 mm, Height of the vessel = 2.1524 m, Clearance from vessel bottom to foundation = 0.75 m, Weight of vessel with contents = 3918.9 kg, Wind pressure = 100 kgf/m<sup>2</sup>, No of brackets = 4, Diameter of bolt circle = 1202.7 mm, Size of base plate for bracket = 150 mm x 150 mm, Height of the C channel from foundation = 2.0264 m, Size of C channel = 150 mm x 75 mm, Area of cross section = 20.88 cm<sup>2</sup>, Modulus of section = 19.4 cm<sup>3</sup>, Radius of gyration = 2.21 cm, MOC for support = IS 800, Max. allowable tensile stress = 1400 kgf/cm<sup>2</sup>, Max. allowable compressive stress = 1233 kgf/cm<sup>2</sup>, Max. allowable bending stress = 1575 kgf/cm<sup>2</sup> **14**

- Q.4** (a) State the industrial applications of different types of storage tanks. **03**  
(b) Why permanent vent is provided in storage tank? State the advantages and disadvantages of external roof tank over internal floating roof tank. **04**  
(c) Discuss the design steps for the selection of rafter, girder and column used for column supported conical roof. **07**

**OR**

- Q.4** (a) State the purpose of providing reinforcement pad for nozzle. **03**  
(b) Discuss about normal and emergency venting system for storage vessel. **04**  
(c) Discuss the mechanical design of tube and tube sheet under internal & external pressure for a shell and tube heat exchanger. **07**
- Q.5** (a) Name the various types of stresses induced in the shell of distillation column. **03**  
(b) When skirt support is used for pressure vessel? State any two advantages of skirt support over bracket support. **04**  
(c) What is tray support? Write a short note on different types of tray supports used for distillation column. **07**

**OR**

- Q.5** Distillation is carried out in a packed tower under vacuum. Determine the thickness and weight of shell based on the following given data. **14**  
Shell ID = 1500 mm, Length of Shell - 15 m, External design pressure - 101.325 kPa, Design temperature - 200°C, Shell and head material - SA 240 GrS type 304, Type of Shell Plate join - Double welded butt joint with 10% radiography, Nos. of packing sections - 5, Height of each packing section - 2.5 m, Top disengaging space - 1.2 m, Type of Packing - 25 mm S.S. pall rings, Weight of Packing per m<sup>3</sup> - 480 kg, % void space - 94%, Density of liquid - 1100 kg/m<sup>3</sup>, Weight of attachment (pipes, ladders & platform) - 150 kg/m, Wind pressure - 981 N/m<sup>2</sup>, Insulation thickness - 50 mm, Density of insulation - 500 kg/m<sup>3</sup>, Density of SA 240 GrS type 304 - 7800 kg/m<sup>3</sup>, Maximum allowable stress of shell or head plate material - 104.12 N/mm<sup>2</sup> (tensile), Modulus of elasticity of plate material -  $2 \times 10^5$  N/mm<sup>2</sup>, Poissons ratio of plate material - 0.3, Neglect the stress created by eccentric load and seismic load.

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