

GUJARAT TECHNOLOGICAL UNIVERSITY**BE - SEMESTER-VII EXAMINATION – SUMMER 2025****Subject Code:3170202****Date:19-05-2025****Subject Name:Automotive Component and system Design****Time:02:30 PM TO 05:30 PM****Total Marks:70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.
4. Simple and non-programmable scientific calculators are allowed.

MARKS

- Q.1**
- | | |
|---|-----------|
| (a) Define brake fade. | 03 |
| (b) Explain the design for manufacturing and design for assembly. | 04 |
| (c) Derive Stribeck's equation. | 07 |

- Q.2**
- | | |
|---|-----------|
| (a) Explain preloading of rolling contact bearing. | 03 |
| (b) Define Dynamic Load Carrying Capacity. | 04 |
| (c) A single-row deep groove ball bearing is subjected to a radial force of 8 kN and a thrust force of 3 kN. The shaft rotates at 1200 rpm. The expected life L_{10h} of the bearing is 20 000 h. The minimum acceptable diameter of the shaft is 75 mm. Select a suitable ball bearing for this application. | 07 |

OR

- | | |
|---|-----------|
| (c) A single-row deep groove ball bearing is subjected to a pure radial force of 3 kN from a shaft that rotates at 600 rpm. The expected life L_{10h} of the bearing is 30 000 h. The minimum acceptable diameter of the shaft is 40 mm. Select a suitable ball bearing for this application. | 07 |
|---|-----------|

- Q.3**
- | | |
|---|-----------|
| (a) State different modes of gear teeth failures. | 03 |
| (b) Define following terms (1) Pitch circle (2) Pressure angle (3) Module (4) Addendum. | 04 |
| (c) It is required to design a pair of spur gears with 20° full-depth involute teeth based on the Lewis equation. The velocity factor is to be used to account for dynamic load. The pinion shaft is connected to a 10 kW, 1440 rpm motor. The starting torque of the motor is 150% of the rated torque. The speed reduction is 4 : 1. The pinion as well as the gear is made of plain carbon steel 40C8 ($S = 600 \text{ N/mm}^2$). The factor of safety can be taken as 1.5. Design the gears, specify their dimensions and suggest suitable surface hardness for the gears. | 07 |

OR

- Q.3**
- | | |
|---|-----------|
| (a) Write advantages and disadvantages of worm gears. | 03 |
| (b) Justify the use of involute profile for gear tooth. | 04 |
| (c) A pair of parallel helical gears consists of a 20 teeth pinion meshing with a 40 teeth gear. The helix angle is 25° and the normal pressure angle is 20° . The normal module is 3 mm. Calculate
(i) the transverse module;
(ii) the transverse pressure angle;
(iii) the axial pitch;
(iv) the pitch circle diameters of the pinion and the gear;
(v) the centre distance; and | 07 |

(vi) the addendum and dedendum circle diameters of the pinion.

- Q.4** (a) Explain the Ackermann steering principle. **03**
(b) Compare Disc Brake with Drum Brake. **04**
(c) Following data is given for a caliper disk brake with annular pad, for the front wheel of the motorcycle: **07**
torque capacity = 1500 N-m
outer radius of pad = 150 mm
inner radius of pad = 100 mm
coefficient of friction = 0.35
average pressure on pad = 2 MPa
number of pads = 2
Calculate the angular dimension of the pad.

OR

- Q.4** (a) Define brake efficiency. **03**
(b) Explain the functions of piston rings. **04**
(c) The following data is given for a caliper disk brake, with circular pad, for the lightweight two-wheeler, **07**
torque capacity = 1500 N-m
number of caliper brakes on the wheel = 3
number of pads on each caliper brake = 2
coefficient of friction = 0.35
average pressure on pad = 2 MPa The ratio of pad radius to the distance of the pad center from axis of disk is 0.2. Calculate the radius of the pad.

- Q.5** (a) Calculate the minimum stopping distance for a vehicle travelling at 60 km/hr with a deceleration equal to the acceleration due to gravity. **03**
(b) Explain tractive effort. **04**
(c) The following data is given for the piston of a four-stroke diesel engine: **07**
Cylinder bore = 250 mm
Maximum gas pressure = 4 MPa
Allowable bearing pressure for skirt = 0.4 MPa
Ratio of side thrust on liner to maximum gas load on piston = 0.1
Width of top land = 45 mm
Width of ring grooves = 6 mm
Total number of piston rings = 4
Axial thickness of piston rings = 7 mm
Calculate:
(i) length of the skirt; and
(ii) length of the piston.

OR

- Q.5** (a) Define the advantages and disadvantages of wet liner and Dry liner in IC Engines. **03**
(b) Explain the desirable properties of cylinder materials. **04**
(c) Determine the dimensions of cross-section of the connecting rod for a diesel engine with the following data: **07**
Cylinder bore = 100 mm
Length of connecting rod = 350 mm
Maximum gas pressure = 4 MPa
Factor of safety = 6
